

## Final Exam

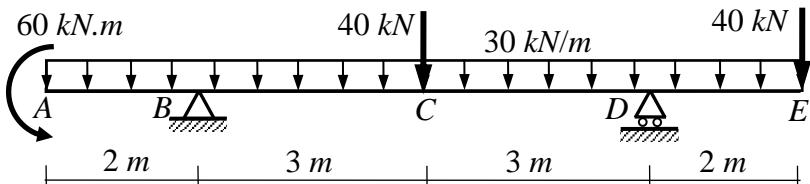
Total Marks: 70

No. of Questions: 50 (Attempt all questions)

### Choose the nearest answer.

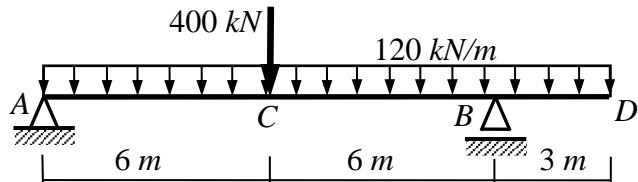
For the shown beam, it is required to determine the slope at **A** and the deflections at **A**, **C** and **E** by using the **double integration method**.

$$EI = 1.0 \times 10^5 \text{ kN.m}^2$$



1. The vertical reaction at the support **D** is:  
(A) 213.3 kN↑   (B) 166.7 kN↑   (C) 156.7 kN↑   (D) 143.3 kN↑
2. The bending moment equation ( $M$ ) in the last part **DE** is:  
(A)  $-60 - 15x^2 + 500(x-2)/3 - 40(x-5) + 500(x-8)/3$   
(C)  $-60x - 15x^2 + 500(x-2)/3 - 40(x-5) + 640(x-8)/3$   
(B)  $-60x^0 - 15x^2 + 500(x-2)/3 - 40(x-5) + 640(x-8)/3$   
(D)  $-60x^0 - 15x^2 + 640(x-2)/3 - 40(x-5) + 500(x-8)/3$
3.  $EIy = \dots$   
(A)  $-60x - 1.25x^4 + 250(x-2)^3/9 - 20(x-5)^3/3 + 320(x-8)^3/9 + C_1x + C_2$   
(C)  $-60 - 1.25x^4 + 320(x-2)^3/9 - 20(x-5)^3/3 + 250(x-8)^3/9 + C_1x + C_2$   
(B)  $-30x^2 - 1.25x^4 + 320(x-2)^3/9 - 20(x-5)^3/3 + 250(x-8)^3/9 + C_1x + C_2$   
(D)  $-30x^2 - 1.25x^4 + 250(x-2)^3/9 - 20(x-5)^3/3 + 320(x-8)^3/9 + C_1x + C_2$
4. Boundary Conditions are:  
(A) At  $x=2, y=0$  & at  $x=8, y=0$    (B) At  $x=0, y'=0$  & at  $x=10, y'=0$    (C) At  $x=2, y=0$  & at  $x=10, y=0$
5.  $C_1 = \dots$ :  
(A) zero   (B) 220   (C) -560   (D) 180
6.  $C_2 = \dots$ :  
(A) -220   (B) -180   (C) -1600   (D) 180
7. The slope at **A**,  $\theta_A = y'_A$  is:  
(A) 0.0018 rad ↗   (B) 0.124 rad ↗   (C) 0.0325 rad ↗   (D) 0.0005 rad ↗
8. The deflection at **A**,  $y_A$  is:  
(A) 10.2 mm↓   (B) 8.8 mm↓   (C) 2.2 mm↓   (D) 6.7 mm↓
9. The deflection at **C**,  $y_C$  is:  
(A) 20.1 mm↓   (B) 13.7 mm↓   (C) 7.1 mm↓   (D) 1 mm↓
10. The deflection at **E**,  $y_E$  is:  
(A) 9.91 mm↓   (B) 13.17 mm↓   (C) 7.11 mm↓   (D) 2.47 mm↓

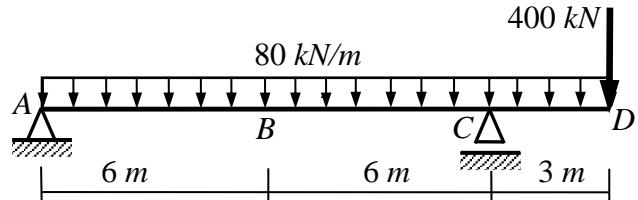
For the shown beam, it is required to determine the slope at **A** and the deflections at **C**&**D** by using the **moment-area method**.  $EI = 1.0 \times 10^6 \text{ kN.m}^2$



11. The vertical reaction at the hinged support **B** is:  
(A) 260 kN↑   (B) 200 kN↑   (C) 875 kN↑   (D) 1325 kN↑
12. The bending moment at **B** is:  
(A) -360 kNm   (B) -960 kNm   (C) -2400 kNm   (D) -540 kNm
13. The bending moment at **C** is:  
(A) 2160 kNm   (B) 2400 kNm   (C) 2625 kNm   (D) 3090 kNm
14. The deviation of **B** relative to the tangent of the elastic curve at **A**,  $t_{B/A}$  is:  
(A) 0.064 m   (B) 0.032 m   (C) 0.134 m   (D) 0.092 m
15. The deviation of **C** relative to the tangent of the elastic curve at **A**,  $t_{C/A}$  is:  
(A) 0.065 m   (B) 0.053 m   (C) 0.025 m   (D) 0.092 m
16. The slope of the tangent of the elastic curve at **A**,  $\theta_A$  is:  
(A) 0.0112 rad ↗   (B) 0.0048 rad ↗   (C) 1.2 rad ↗   (D) 1.2 rad ↗
17. The deviation of **D** relative to the tangent of the elastic curve at **A**,  $t_{D/A}$  is:  
(A) 0.104 m   (B) 0.048 m   (C) 0.196 m   (D) 0.072 m
18. The deflection at **C**,  $\delta_C$  is:  
(A) 41.9 mm↓   (B) 8.3 mm↓   (C) 24.5 mm↓   (D) 18.3 mm↓
19. The deflection at **D**,  $\delta_D$  is:  
(A) 18.3 mm↑   (B) 29.0 mm↓   (C) 18.3 mm↓   (D) 29.0 mm↑
20. The nearest elastic curve of the shown beam is:  
(A)   
(B)   
(C)   
(D)

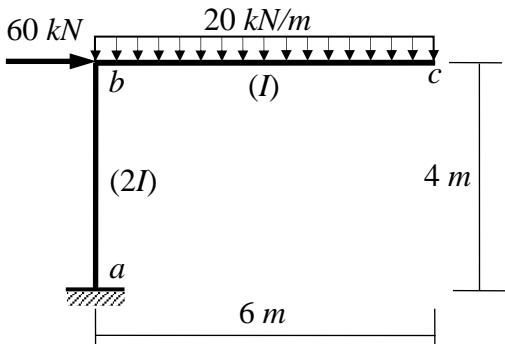


For the shown beam, it is required to determine the slopes at **A&D**, the deflections at **B&D** and the maximum deflection for the beam by using the **conjugate beam method**.  $EI = 1.0 \times 10^6 \text{ kN.m}^2$ .



21. After loading  $M$  on the conjugate beam, the elastic reaction at support **A** is:  
(A)  $2640 \text{ kN.m}^2 \uparrow$    (B)  $1208 \text{ kN.m}^2 \uparrow$    (C)  $604 \text{ kN.m}^2 \downarrow$    (D)  $1320 \text{ kN.m}^2 \uparrow$
22. After loading  $M$  on the conjugate beam, the elastic reaction at support **D** is:  
(A)  $1208 \text{ kN.m}^2 \uparrow$    (B)  $1320 \text{ kN.m}^2 \downarrow$    (C)  $604 \text{ kN.m}^2 \uparrow$    (D)  $2640 \text{ kN.m}^2 \downarrow$
23. The slope of the tangent of the elastic curve at **A**,  $\theta_A$  is:  
(A)  $0.006 \text{ rad } \circlearrowleft$    (B)  $0.0013 \text{ rad } \circlearrowleft$    (C)  $0.0026 \text{ rad } \circlearrowleft$    (D)  $0.0006 \text{ rad } \circlearrowleft$
24. The slope of the tangent of the elastic curve at **D**,  $\theta_D$  is:  
(A)  $0.0013 \text{ rad } \circlearrowleft$    (B)  $0.006 \text{ rad } \circlearrowleft$    (C)  $0.0026 \text{ rad } \circlearrowleft$    (D)  $0.0006 \text{ rad } \circlearrowleft$
25. The deflection at **B**,  $\delta_B$  is:  
(A)  $4.27 \text{ mm} \uparrow$    (B)  $17.56 \text{ mm} \uparrow$    (C)  $3.78 \text{ mm} \downarrow$    (D)  $7.56 \text{ mm} \downarrow$
26. The deflection at **D**,  $\delta_D$  is:  
(A)  $14.27 \text{ mm} \uparrow$    (B)  $8.53 \text{ mm} \uparrow$    (C)  $5.85 \text{ mm} \downarrow$    (D)  $2.93 \text{ mm} \downarrow$
27. The maximum downward deflection is at a distance from support **A** = ....:  
(A)  $4.91 \text{ m}$    (B)  $6.00 \text{ m}$    (C)  $9.82 \text{ m}$    (D)  $2.99 \text{ m}$
28. The maximum downward deflection is:  
(A)  $4.0 \text{ mm}$    (B)  $8.0 \text{ mm}$    (C)  $6.5 \text{ mm}$    (D)  $12.1 \text{ mm}$
29. The nearest elastic curve of the shown beam is:  
(A)   (B)   (C)   (D)

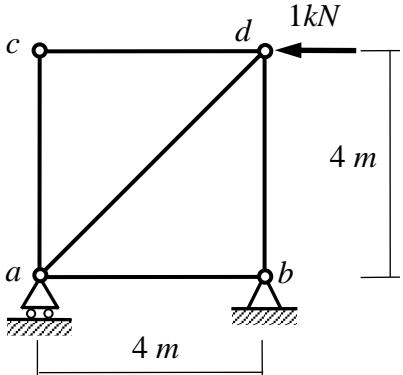
For the shown frame, it is required to determine the horizontal and vertical displacements at **c** ( $\delta_{ch}$  and  $\delta_{cv}$ ) and the slope at **c** ( $\theta_c$ ) using the **virtual work method**. The relative moments of inertia are given between brackets.  $EI=1.0 \times 10^6 \text{ kN.m}^2$ .



30. The bending moment at **b** due to the given load is:  
(A)  $-20 \text{ kN.m}$    (B)  $-160 \text{ kN.m}$    (C)  $-360 \text{ kN.m}$    (D)  $-240 \text{ kN.m}$
31. The bending moment at **a** due to the given load is:  
(A)  $-600 \text{ kN.m}$    (B)  $-60 \text{ kN.m}$    (C)  $-240 \text{ kN.m}$    (D)  $-360 \text{ kN.m}$
32. The value of the bending moment at **b** due to unit vertical load at **c** is:  
(A)  $4 \text{ kN.m}$    (B)  $1 \text{ kN.m}$    (C)  $6 \text{ kN.m}$    (D)  $24 \text{ kN.m}$
33. The horizontal displacement at **c**,  $\delta_{ch}$  is:  
(A)  $8.1 \text{ mm} \rightarrow$    (B)  $3.2 \text{ mm} \leftarrow$    (C)  $0.1 \text{ mm} \rightarrow$    (D)  $2.1 \text{ mm} \rightarrow$
34. The vertical displacement at **c**,  $\delta_{cv}$  is:  
(A)  $5 \text{ mm} \uparrow$    (B)  $9 \text{ mm} \downarrow$    (C)  $26 \text{ mm} \downarrow$    (D)  $2 \text{ mm} \downarrow$
35. The slope at **c**,  $\theta_c$  is:  
(A)  $0.0097 \text{ rad } \circlearrowleft$    (B)  $0.0017 \text{ rad } \circlearrowleft$    (C)  $0.0072 \text{ rad } \circlearrowleft$    (D)  $0.0172 \text{ rad } \circlearrowleft$

For the shown truss, it is required to determine the horizontal and vertical displacement at **d** ( $\delta_{dh}$  and  $\delta_{dv}$ ) using the **virtual work method**.  $EA=1000 \text{ kN}$

36. The force in member **cd** due to the given load ( $N_o$ ) is:  
(A)  $-1.4 \text{ kN}$    (B)  $-1 \text{ kN}$    (C) zero   (D)  $1 \text{ kN}$
37. The force in member **bd** due to the given load ( $N_o$ ) is:  
(A) zero   (B)  $1 \text{ kN}$    (C)  $-1 \text{ kN}$    (D)  $-1.4 \text{ kN}$
38. The value of the force in member **bd** due to vertical unit load at **d** is:  
(A) zero   (B)  $1 \text{ kN}$    (C)  $0.5 \text{ kN}$    (D)  $1.4 \text{ kN}$
39. The horizontal displacement at **d**,  $\delta_{dh}$  is:  
(A)  $12.1 \text{ mm} \rightarrow$    (B)  $9.3 \text{ mm} \leftarrow$    (C)  $3.2 \text{ mm} \leftarrow$    (D)  $19.3 \text{ mm} \leftarrow$
40. The vertical displacement at **d**,  $\delta_{dv}$  is:  
(A)  $4 \text{ mm} \uparrow$    (B)  $2 \text{ mm} \uparrow$    (C) zero   (D)  $20 \text{ mm} \downarrow$



For the shown frames in **1** and **2** and beam in **3**, it is required to draw the influence line (*I.L.*) for some functions (reaction - shear force - bending moment).

41. The diagram shown in **A** is the *I.L.* for:

- (A)  $A_y$  of the frame **1**.
- (B)  $A_y$  of the frame **2**.
- (C)  $A_y$  of the beam **3**.**
- (D) shear force at **A** of the frame **2**.

42. The diagram shown in **B** is the *I.L.* for:

- (A)  $E_y$  of the frame **1**.
- (B)  $E_y$  of the frame **2**.**
- (C)  $B_y$  of the beam **3**.
- (D)  $C_y$  of the beam **3**.

43. The diagram shown in **C** is the *I.L.* for:

- (A)  $A_x$  of the frame **1**.**
- (B)  $A_x$  of the frame **2**.
- (C) Bending moment at **C<sub>right</sub>** of the frame **2**.
- (D)  $B_y$  of the beam **3**.

44. The diagram shown in **D** is the *I.L.* for:

- (A)  $A_y$  of the frame **1**.
- (B) Shear force at **E** of the frame **1**.
- (C)  $A_y$  of the beam **3**.
- (D) Shear force at **E** of the beam **3**.**

45. The diagram shown in **E** is the *I.L.* for:

- (A)  $A_y$  of the frame **1**.
- (B)  $A_y$  of the frame **2**.**
- (C) Shear force at **E** of the frame **2**.
- (D)  $A_y$  of the beam **3**.

46. The diagram shown in **F** is the *I.L.* for:

- (A)  $A_y$  of the frame **1**.**
- (B)  $A_x$  of the frame **2**.
- (C)  $A_y$  of the frame **2**.
- (D)  $A_x$  of the beam **3**.

47. The diagram shown in **G** is the *I.L.* for:

- (A) Bending moment at **C<sub>right</sub>** of the frame **2**.**
- (B) Shear force at **C<sub>right</sub>** of the frame **2**.
- (C)  $C_y$  of the beam **3**.
- (D) Bending moment at **D** of the beam **3**.

48. The diagram shown in **H** is the *I.L.* for:

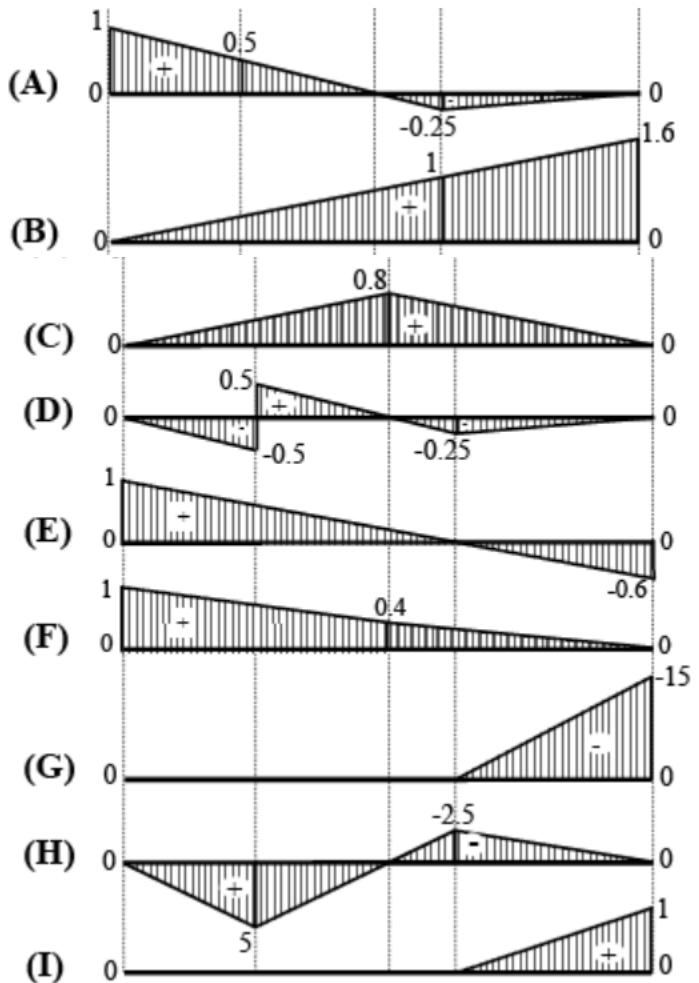
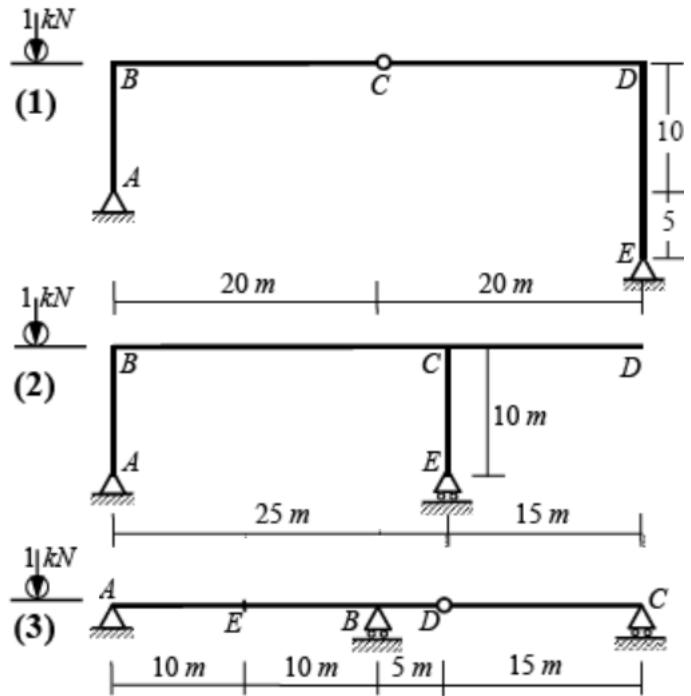
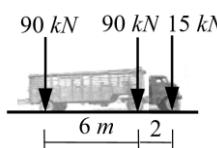
- (A)  $A_y$  of the frame **1**.
- (B) Shear force at **E** of the beam **3**.
- (C)  $A_y$  of the beam **3**.
- (D) Bending moment at **E** of the beam **3**.**

49. The diagram shown in **I** is the *I.L.* for:

- (A) Bending moment at **C<sub>right</sub>** of the frame **2**.
- (B) Shear force at **C<sub>right</sub>** of the frame **2**.
- (C) Bending moment at **D** of the beam **3**.
- (D)  $C_y$  of the beam **3**.**

50. The maximum  $A_y$  of the frame **2** caused by the shown moving truck is:

- (A) 168.6 kN.**
- (B) 195 kN.
- (C) 285.6 kN.
- (D) 390.1 kN.



With my best wishes

Dr. M. Abdel-Kader