

### Answer of Mid-Term Exam

Total Marks: 20

No. of Questions: 2 (Attempt all questions)

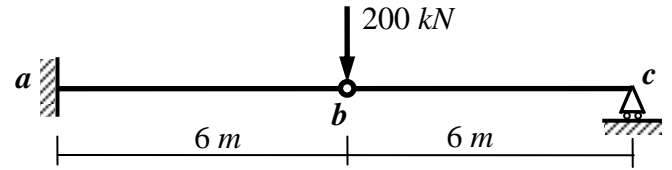
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**Question (1): (10 Marks)**

Using the **double integration method**, determine the deflection at **b**, the slope at section just to the left of **b**, and sketch the elastic curve of the beam.

$EI = 6 \times 10^5 \text{ kN.m}^2$



**Solution:**

- First, determine the reactions as shown.
- Note that all the required ( $y_b$  and  $y'_{bleft}$ ) are in the region  $a-b$  and no reaction comes from part  $b-c$  to part  $a-b$ , so only part  $a-b$  should be studied.

**For region a-b**

$EIy'' = M$

$EIy'' = 200x - 1200$

$EIy' = 100x^2 - 1200x + C_1$

$EIy = \frac{100}{3}x^3 - 600x^2 + C_1x + C_2$

**Boundary Conditions:**

At  $x = 0, y' = 0 \rightarrow C_1 = 0$

At  $x = 0, y = 0 \rightarrow C_2 = 0$

So, the general equation of the deflection  $y$  and slope  $y' (\theta)$  at any distance  $x$  is,

$EIy = \frac{100}{3}x^3 - 600x^2$

$EIy' = 100x^2 - 1200x$

The deflection at **b** is at  $x = 6 \text{ m}$

$EI\delta_b = \frac{100}{3}(6)^3 - 600(6)^2 = -14400$

$\delta_b = -14400/600000 = -0.024 \text{ m}$

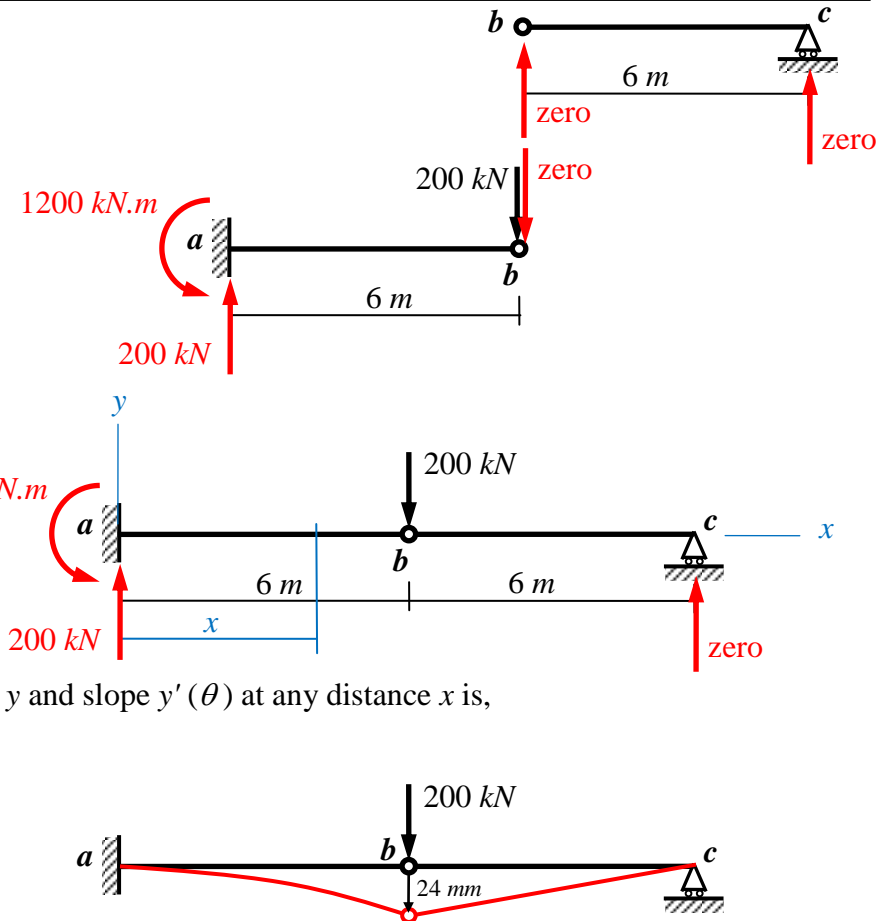
$\delta_b = 24 \text{ mm} \downarrow$

The slope at section just to the left of **b** is at  $x = 6 \text{ m}$

$EIy' = 100(6)^2 - 1200(6) = -3600$

$\theta_b = -3600/600000 = -0.006 \text{ rad}$

$\theta_b = 0.344^\circ \curvearrowright$



Elastic curve

With my best wishes

Dr. M. Abdel-Kader

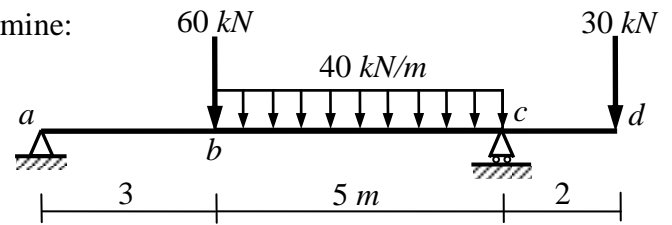
**Question (2): (10 Marks)**

For the shown beam, using the **moment-area method**, determine:

- (a) The slope at  $a$
- (b) The deflection at  $b$
- (c) The deflection at  $d$

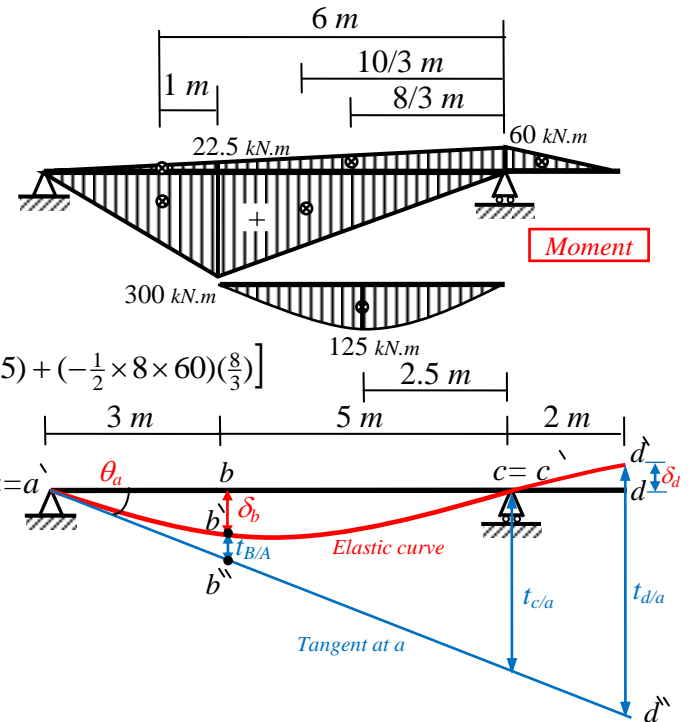
and sketch the elastic curve of the beam.

$$EI = 60 \text{ MN.m}^2$$



**Solution:**

The bending moment diagram may be drawn as shown.



**(a) The slope at a**

$$\theta_a = \frac{t_{c/a}}{8}$$

Apply the second moment-area theorem, then

$$t_{c/a} = \frac{1}{EI} [Area_{ac} \cdot \bar{X}_c]$$

$$= \frac{1}{EI} \left[ \left(\frac{1}{2} \times 3 \times 300\right)(6) + \left(\frac{1}{2} \times 5 \times 300\right)\left(\frac{10}{3}\right) + \left(\frac{2}{3} \times 5 \times 125\right)(2.5) + \left(-\frac{1}{2} \times 8 \times 60\right)\left(\frac{8}{3}\right) \right]$$

$$= \frac{16805}{3EI} = \frac{16805}{3 \times 60000} = 3351/36000 = 0.09336 \text{ m}$$

$$\therefore \theta_a = \frac{t_{c/a}}{8} = \frac{0.09336}{8} = 0.01167 \text{ rad} = 0.6686^\circ$$

$$\theta_a = 0.67^\circ \curvearrowright$$

**(b) The deflection at b**

The deflection at  $b = \delta_b = bb'' - b'b'' = (3/8) t_{c/a} - t_{b/a}$

Applying the second moment-area theorem, then

$$t_{b/a} = \frac{1}{EI} [\text{First moment of area of M - diagram between } a \text{ and } b \text{ about } b]$$

$$= \frac{1}{EI} [Area_{ab} \cdot \bar{X}_b] = \frac{1}{EI} \left[ \left(\frac{1}{2} \times 3 \times 300\right)(1) + \left(-\frac{1}{2} \times 3 \times 22.5\right)(1) \right] = \frac{1665}{4EI} = \frac{1665}{4 \times 60000} = 0.0069375 \text{ m}$$

$$\therefore \delta_b = bb'' - b'b'' = (3/8) t_{c/a} - t_{b/a} = (3/8)(0.09336) - 0.0069375 = 0.0280725 \text{ m} \quad \delta_b = 28.1 \text{ mm} \downarrow$$

**(c) The deflection at d**

$$t_{d/a} = \frac{1}{EI} [Area_{ad} \cdot \bar{X}_d]$$

$$= \frac{1}{EI} \left[ \left(\frac{1}{2} \times 3 \times 300\right)(8) + \left(\frac{1}{2} \times 5 \times 300\right)\left(\frac{16}{3}\right) + \left(\frac{2}{3} \times 5 \times 125\right)(4.5) + \left(-\frac{1}{2} \times 8 \times 60\right)\left(\frac{14}{3}\right) + \left(-\frac{1}{2} \times 2 \times 60\right)\left(\frac{4}{3}\right) \right]$$

$$= \frac{8275}{EI} = \frac{8275}{60000} = 331/2400 = 0.1379167 \text{ m}$$

$$\therefore \delta_d = d'd'' - dd'' = t_{d/a} - (10/8) t_{c/a} = 0.1379167 - (10/8)(0.09336) = 0.021217 \text{ m} \quad \delta_d = 21.2 \text{ mm} \uparrow$$

