

Second Semester Final Examination

- Attempt all questions.
- The Exam consists of **4** questions in **1** page.
- Maximum grade is **60 Marks**.

Question (1): (15 Marks)

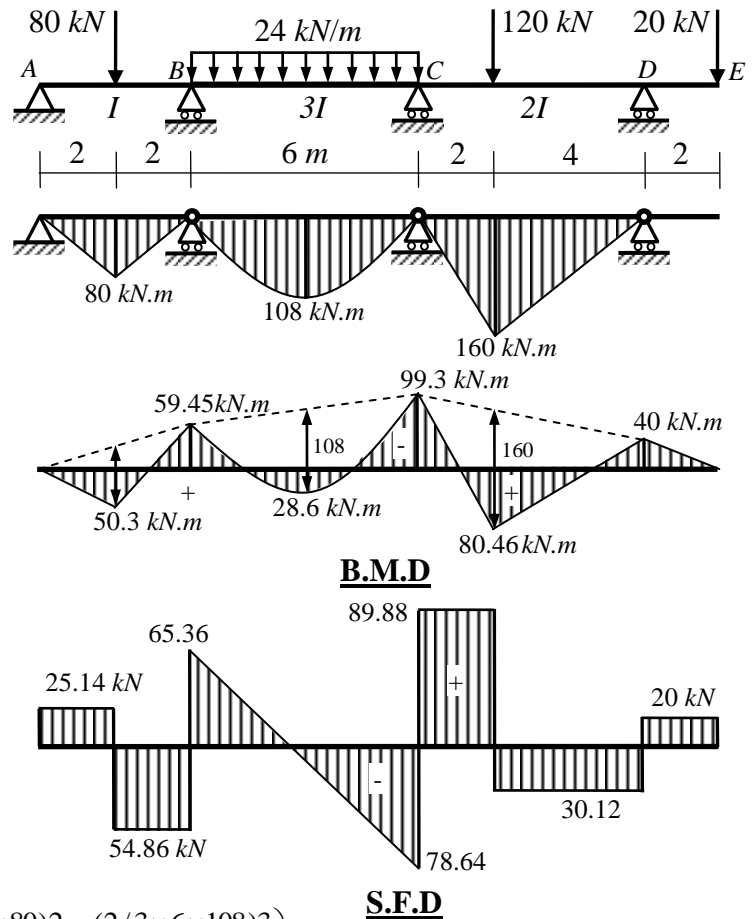
Using the three-moments equation, draw the shear force and bending moment diagrams for the shown continuous beam of variable moment of inertia.

Solution:

The simply supported moment diagram on *AB*, *BC* and *CD* is as shown.

Since the support *A* is simply supported, $M_A = 0$. The moment at *D* is

$$M_D = -20(2) = -40 \text{ kN.m.}$$



Applying three-moments equation for spans *AB* and *BC*:

$$M_A \left(\frac{4}{I} \right) + 2M_B \left(\frac{4}{I} + \frac{6}{3I} \right) + M_C \left(\frac{6}{3I} \right) = -6 \left(\frac{(0.5 \times 4 \times 80)2}{4I} + \frac{(2/3 \times 6 \times 108)3}{6 \times 3I} \right)$$

$$\therefore 6M_B + M_C = -456 \quad (1)$$

For spans *BC* and *CD* : ($M_D = -40 \text{ kN.m}$)

$$M_B \left(\frac{6}{3I} \right) + 2M_C \left(\frac{6}{3I} + \frac{6}{2I} \right) + M_D \left(\frac{6}{2I} \right) = -6 \left(\frac{(2/3 \times 6 \times 108)3}{6 \times 3I} + \frac{(0.5 \times 4 \times 160)2/3 \times 4 + (0.5 \times 2 \times 160)(4 + 2/3)}{6 \times 2I} \right)$$

or

$$M_B \left(\frac{6}{3I} \right) + 2M_C \left(\frac{6}{3I} + \frac{6}{2I} \right) + M_D \left(\frac{6}{2I} \right) = -6 \left(\frac{(2/3 \times 6 \times 108)3}{6 \times 3I} + \frac{(0.5 \times 6 \times 160)(6 + 4)/3}{6 \times 2I} \right)$$

$$\therefore M_B + 5M_C = -556 \quad (2)$$

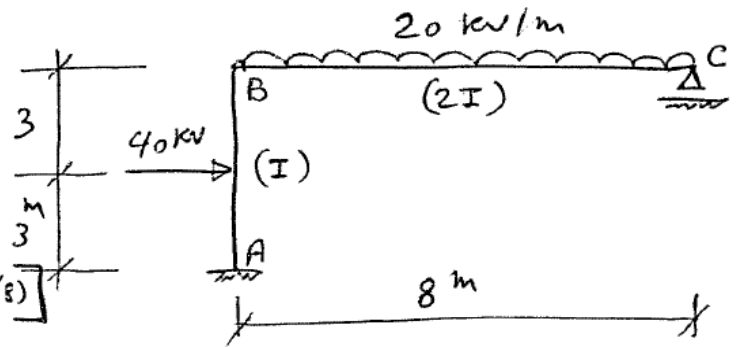
Solving Eqs. (1) and (2) yields $M_B = -59.448 = -59.45 \text{ kN.m}$ and $M_C = -99.3 \text{ kN.m}$.

$$R_A = 25.14 \text{ kN}, \quad R_B = 120.22 \text{ kN}, \quad R_C = 168.52 \text{ kN} \quad \text{and} \quad R_D = 50.12 \text{ kN}.$$

The bending moment and shear force diagram are shown above.

Question (2): (15 Marks)

For the shown frame with variable moment of inertia, using the virtual work method, find the reactions at the supports A and C. The relative moments of inertia are given between brackets and E is constant.



$$\delta_{10} = \int \frac{M_0 M_1}{EI} dl$$

$$= \frac{1}{EI} \left[(-6 \times 640)(8) + \left(-\frac{1}{2} \times 3 \times 120\right)(8) \right]$$

$$+ \frac{1}{2EI} \left[\left(-\frac{1}{3} \times 8 \times 640\right) \left(\frac{3}{4} \times 8\right) \right]$$

$$= -\frac{32160}{EI} - \frac{5120}{EI}$$

$$\delta_{10} = -\frac{37280}{EI}$$

$$\delta_{11} = \int \frac{M_1 M_1}{EI} dl$$

$$= \frac{1}{EI} \left[(6 \times 8)(8) \right] + \frac{1}{2EI} \left[\left(\frac{1}{2} \times 8 \times 8\right) \left(\frac{2}{3} \times 8\right) \right]$$

$$\delta_{11} = \frac{469.3333}{EI}$$

$$\delta_{10} + X_1 \delta_{11} = 0$$

$$\therefore X_1 = -\frac{-37280}{469.3333} = 79.43 \text{ kN}$$

$$R_y = X_1 = 79.43 \text{ kN } \uparrow$$

$$\sum F_y = 0$$

$$A_y + C_y - 20 \times 8 = 0$$

$$A_y = 80.57 \text{ kN } \uparrow$$

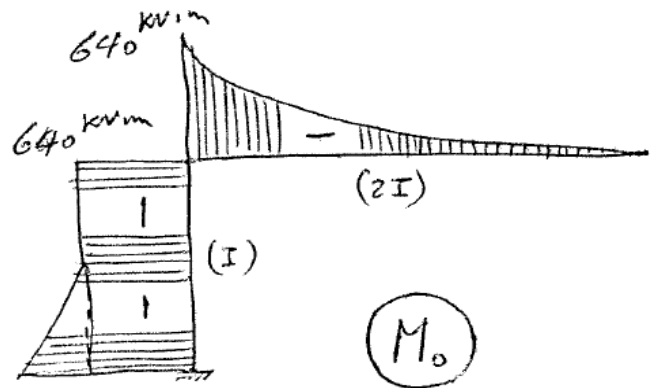
$$\sum F_x = 0$$

$$A_x = 40 \text{ kN } \leftarrow$$

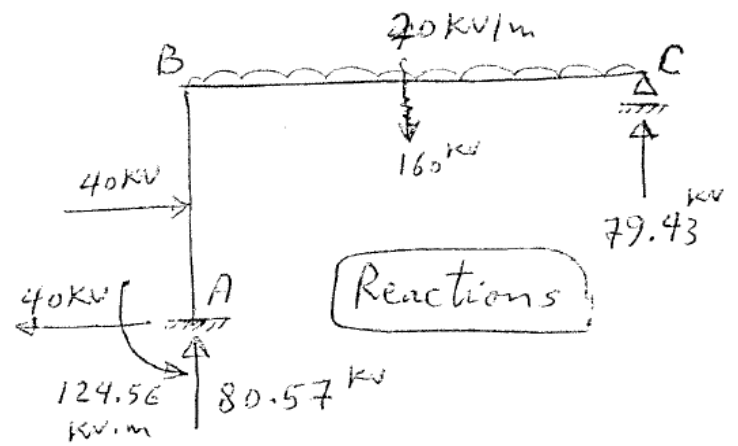
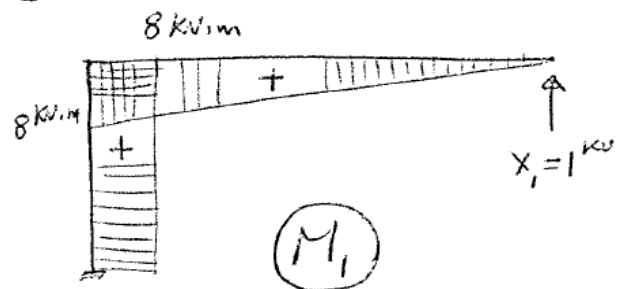
$$\sum M_A = 0$$

$$-M_A - 79.43(8) + (20 \times 8)(4) + 40(3) = 0$$

$$M_A = 124.56 \text{ kN.m } \curvearrowright$$



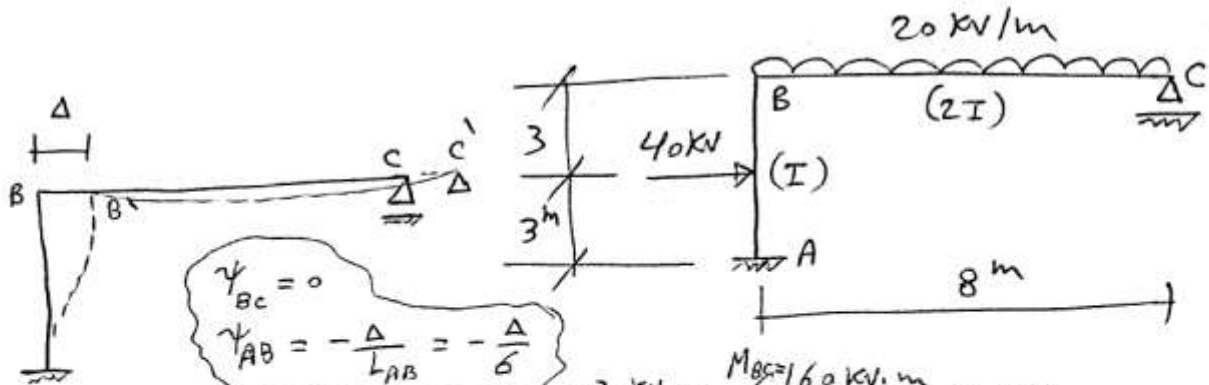
760 kN.m



Question (3): (15 Marks)

For the frame of Question (2), using the slope deflection method,

- (a) Find the rotation at B (θ_B) and the sway of the frame Δ .
- (b) Draw the bending moment diagram.



$\psi_{BC} = 0$
 $\psi_{AB} = -\frac{\Delta}{L_{AB}} = -\frac{\Delta}{6}$

* Unknowns: θ_B and Δ

* Equilibrium equations

$$\sum M_B = M_{BA} + M_{BC} = 0$$

$$\sum F_x = 40 + A_x = 0$$

$$M_{BA} = M_{BA}^F + \frac{2EI}{L}(2\theta_B + \theta_A + 3\psi_{AB})$$

$$= 30 + \frac{2EI}{6}(2\theta_B + 0 - 3\frac{\Delta}{6}) = 30 + EI(\frac{2}{3}\theta_B - \frac{\Delta}{6})$$

$$M_{BC} = M_{BC}^F + \frac{3E(2I)}{L}(\theta_B + \psi_{BC}) = -160 + EI(\frac{3}{4}\theta_B)$$

$$\sum M_B = -130 + EI(\frac{17}{12}\theta_B - \frac{\Delta}{6}) = 0$$

$$\frac{17}{12}\theta_B - \frac{\Delta}{6} = \frac{130}{EI} \quad (1)$$

$$\sum F_x = 40 + \frac{M_{AB} + M_{BA}}{6} - 20 = 0$$

But $M_{AB} = -30 + \frac{2EI}{6}(\theta_B - \frac{\Delta}{6}) = -30 + EI(\frac{\theta_B}{3} - \frac{\Delta}{6})$

$$\sum F_x = 20 + \frac{1}{6}[-30 + EI(\frac{\theta_B}{3} - \frac{\Delta}{6}) + 30 + EI(\frac{2}{3}\theta_B - \frac{\Delta}{6})]$$

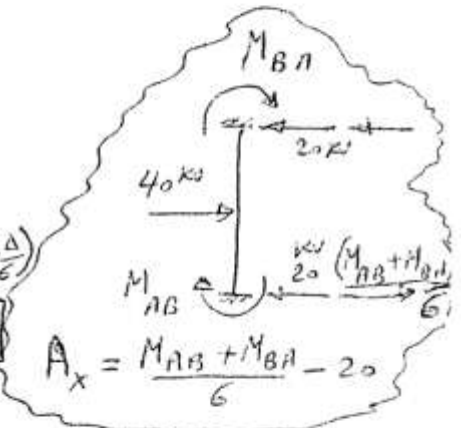
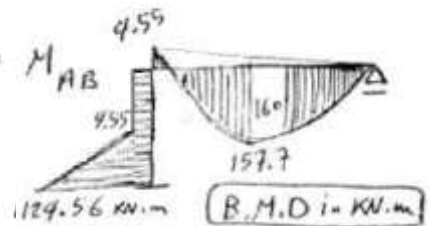
$$= 20 + \frac{EI}{6}(\theta_B - \frac{\Delta}{3})$$

$$\theta_B - \frac{\Delta}{3} = \frac{120}{EI} \quad (2)$$

$$\frac{17}{6}\theta_B - \frac{\Delta}{3} = \frac{260}{EI} \quad (1')$$

$$\theta_B = \frac{2280}{11EI} = \frac{207.2727}{EI}$$

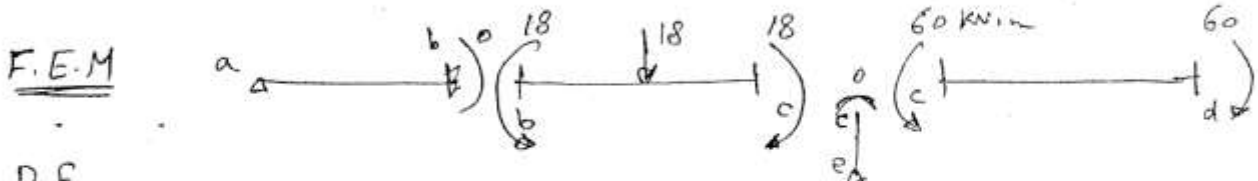
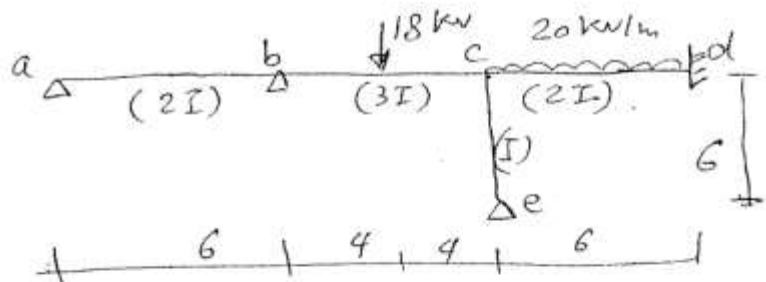
$$\Delta = \frac{981.82}{EI} = \frac{10800}{11EI}$$



$M_{AB} = 124.56 \text{ kN.m}$
 $M_{BA} = 4.55 \text{ kN.m}$
 $M_{BC} = 4.55 \text{ kN.m}$

Question (4): (15 Marks)

For the shown frame with variable moment of inertia, **using the moment distribution method**, draw the bending moment diagram. E is constant. The relative moments of inertia are given between brackets.



D.F.

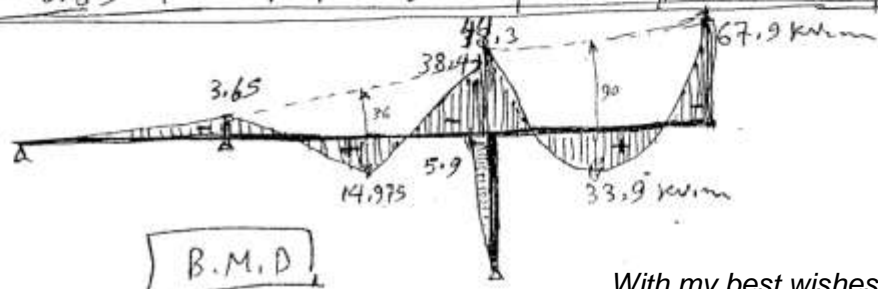
Joint b $K_{ba} = \frac{3E(2I)}{6} = EI$ $K_{bc} = \frac{4E(3I)}{8} = 1.5EI$

$D.F_{ba} = 0.4$ $D.F_{bc} = 0.6$

Joint c $K_{cb} = 1.5EI$ $K_{ce} = \frac{3EI}{6} = 0.5EI$ $K_{cd} = \frac{4E(2I)}{6} = \frac{4}{3}EI$

$D.F_{cb} = 0.45$ $D.F_{ce} = 0.15$ $D.F_{cd} = 0.4$

Joint	b		c			d
Member	ba	bc	cb	ce	cd	dc
D.F	0.4	0.6	0.45	0.15	0.4	1
FEM	0	-18	18	0	-60	60
B.M			18.9	6.3	16.8	
C.O.M		9.45				8.4
B.M	3.42	5.13				
C.O.M			2.565			
B.M			-1.15425	-0.38475	-1.026	
C.O.M		-0.5771				-0.513
B.M	0.231	0.3463				
C.O.F			0.173			
B.M			-0.078	-0.026	-0.07	
M	3.65	-3.65	38.4	5.9	-44.3	67.9



With my best wishes
Dr. M. Abdel-Kader