

### Answer of Final Exam

Total Marks: **90**

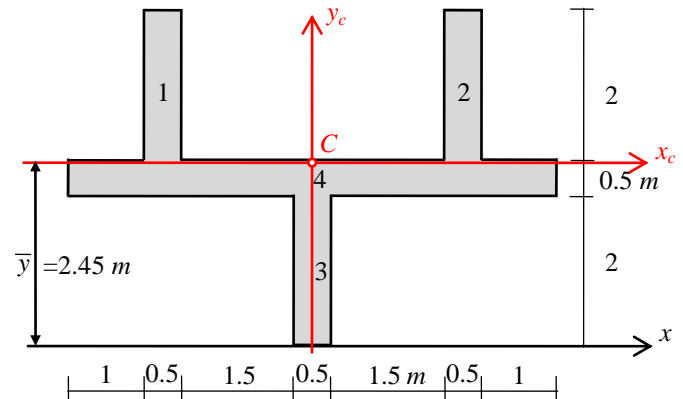
No. of Questions: **3** (Attempt all questions)

#### Question (1): (15 Marks)

(a) For the shown cross-section, determine the following:

- The location of the centroid.
- The moments of inertia about the centroidal axes.
- The polar moment of inertia.
- The radii of gyration ( $r_x$  &  $r_y$ ).

**Note:** Divide the cross-section to 4 elements as shown on the figure.



Element	$A_i$	$y_i$	$A_i y_i$	$I_{xi}$	$dy_i = y_i - \bar{y}$	$A_i (dy_i)^2$	$I_{yi}$	$dx_i$	$A_i (dx_i)^2$
1	$0.5 \times 2$	3.5	3.5	1/3	1.05	1.1025	1/48	-2	4
2	$0.5 \times 2$	3.5	3.5	1/3	1.05	1.1025	1/48	2	4
3	$0.5 \times 2$	1	1	1/3	-1.45	2.1025	1/48	0	0
4	$0.5 \times 6.5$	2.25	7.3125	13/192	-0.2	0.13	2197/192	0	0
$\Sigma$	6.25	-	15.3125	$205/192 = 1.0677$	-	4.4375	$2209/192 = 11.5052$		8

(a) Location of the centroid:

- From symmetry  $y_c$ -axis is as shown

$$\bar{y} = \frac{\sum A_i y_i}{\sum A_i} = \frac{15.3125}{6.25} = \boxed{2.45 \text{ m}}$$

(b) The moments of inertia about the centroidal axes

$$I_{x_c} = \sum (I_{x_i} + A_i (dy_i)^2) = \sum I_{x_i} + \sum A_i (dy_i)^2 = 1.0677 + 4.4375 = 5.5052 = \boxed{5.5 \text{ m}^4}$$

$$I_{y_c} = \sum (I_{y_i} + A_i (dx_i)^2) = \sum I_{y_i} + \sum A_i (dx_i)^2 = 11.5052 + 8 = 19.5052 = \boxed{19.5 \text{ m}^4}$$

(c) The polar moment of inertia ( $I_p$ ) =  $I_{x_c} + I_{y_c} = 5.5 + 19.5 = \boxed{25.0 \text{ m}^4}$

(d)  $r_x = \sqrt{I_{x_c} / A} = \sqrt{5.5 / 6.25} = \boxed{0.938 \text{ m}}$  and  $r_y = \sqrt{I_{y_c} / A} = \sqrt{19.5 / 6.25} = \boxed{1.766 \text{ m}}$

**Question (1): (15 Marks)**

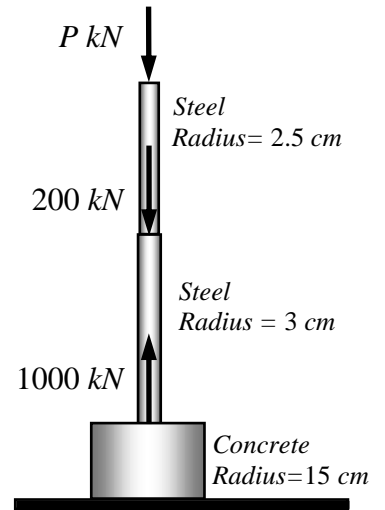
(b) A column of variable circular cross-section is subjected to axial loads as shown. Determine the safe range of  $P$ .

**Given Data:**

For **Steel**: Allowable compressive and tensile stresses =  $140 \text{ MPa}$

For **Concrete**: Allowable compressive stress =  $80 \text{ MPa}$

Allowable tensile stress =  $10 \text{ MPa}$



**Solution:**

Allowable **tensile** stress for concrete =  $10 \text{ N/mm}^2$   
 Allowable **compressive** stress for steel =  $140 \text{ N/mm}^2$

**For Steel (Upper part):**

$$\sigma_{Steel} = \frac{P_{Steel}}{A_{Steel}} = \frac{P \times 10^3}{\pi(25)^2} \leq 140 \text{ N/mm}^2$$

$$\rightarrow P \leq 140 \times \pi(25)^2 / 1000$$

$$\rightarrow P \leq 274.89 \text{ kN} \quad \dots (1)$$

**For Steel (Lower part):**

$$\sigma_{Steel} = \frac{P_{Steel}}{A_{Steel}} = \frac{(P + 200) \times 10^3}{\pi(30)^2} \leq 140 \text{ N/mm}^2$$

$$\rightarrow P + 200 \leq 140 \times \pi(30)^2 / 1000$$

$$\rightarrow P + 200 \leq 395.841$$

$$\rightarrow P \leq 195.84 \text{ kN} \quad \dots (2)$$

**For Concrete:**

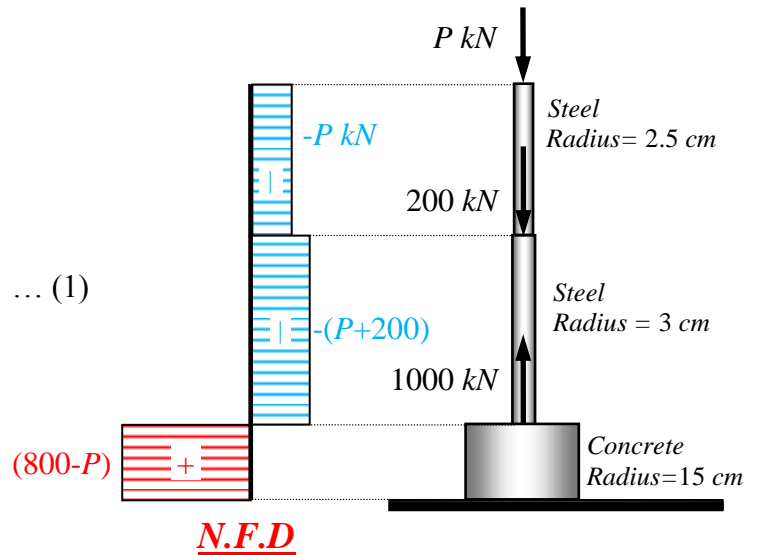
$$\sigma_{Con} = \frac{P_{Con}}{A_{Con}} = \frac{(800 - P) \times 10^3}{\pi(150)^2} \leq 10 \text{ N/mm}^2$$

$$\rightarrow 800 - P \leq 10 \times \pi(150)^2 / 1000$$

$$\rightarrow 800 - P \leq 706.86$$

$$\rightarrow -P \leq -93.14$$

$$\rightarrow P \geq 93.14 \text{ kN} \quad \dots (3)$$



**$\therefore$  From (1), (2) and (3), the safe range of axial force  $P$  is (93.14 kN to 195.84 kN)**

With my best wishes

**Dr. M. Abdel-Kader**

**Question (2): (15 Marks)**

(a) For the shown column, draw the normal stress distribution at the base section ( $40 \text{ cm} \times 60 \text{ cm}$ ) and calculate the maximum normal stresses. Neglect the column weight.

**Solution:**

$$N = -500 - 200 = -700 \text{ kN}$$

$$M_x = +500 \times 30 + 200 \times 30 = 21000 \text{ kN.cm}$$

$$M_y = +500 \times 20 - 200 \times 100 - 100 \times 300 = -40000 \text{ kN.cm}$$

$$A = 40 \times 60 = 2400 \text{ cm}^2$$

$$I_x = 40 \times 60^3 / 12 = 720000 \text{ cm}^4$$

$$I_y = 60 \times 40^3 / 12 = 320000 \text{ cm}^4$$

$$\sigma = \pm \frac{N}{A} \pm \frac{M_x}{I_x} y \pm \frac{M_y}{I_y} x = -\frac{700}{2400} + \frac{21000}{720000} y - \frac{40000}{320000} x$$

N.A.

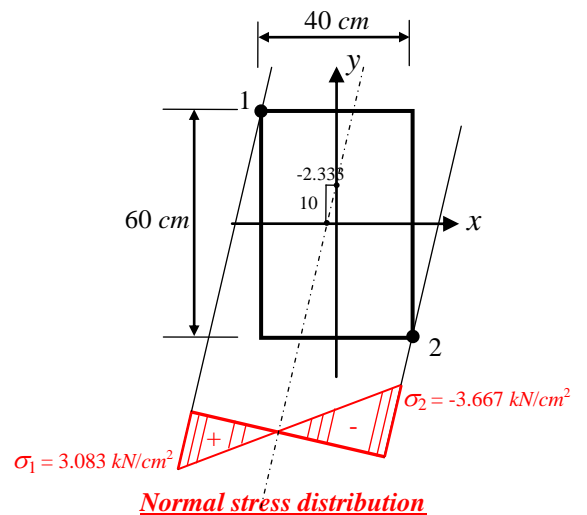
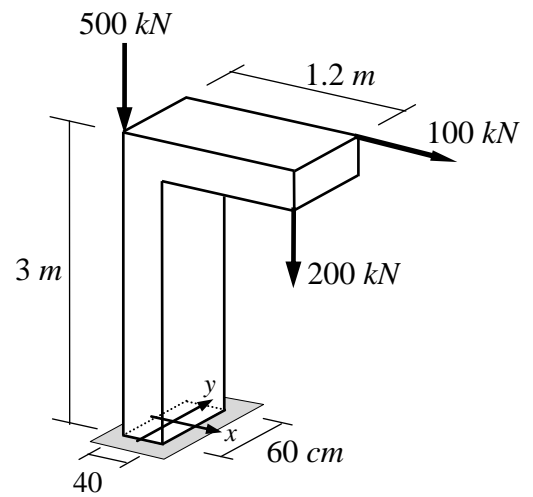
$$-\frac{7}{24} + \frac{7}{240} y - \frac{1}{8} x = 0$$

$$\text{At } x = 0; \quad y = +\frac{7 \times 240}{24 \times 7} = 10 \text{ cm} \quad \rightarrow (0, 10)$$

$$\text{At } y = 0; \quad x = -\frac{7 \times 8}{24 \times 1} = -7/3 = -2.333 \text{ cm} \quad \rightarrow (-2.333, 0)$$

$$\sigma_1 = -\frac{7}{24} + \frac{7}{240}(30) - \frac{1}{8}(-20) = 37/12 = \boxed{+3.083 \text{ kN/cm}^2}$$

$$\sigma_2 = -\frac{7}{24} + \frac{7}{240}(-30) - \frac{1}{8}(20) = -11/3 = \boxed{-3.667 \text{ kN/cm}^2}$$



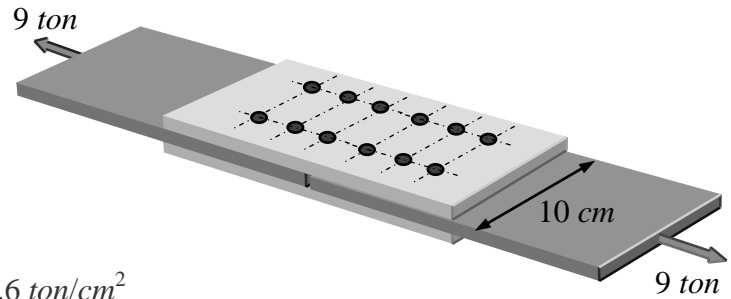
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### Question (2): (15 Marks)

(b) For the shown bolted butt joint, determine the safe diameter of bolts. The width of plates is 10 cm, and the thickness of plates is 12 mm. The allowable stresses are as follows:

**Bolts:**  $\tau_{all} = 1.1 \text{ ton/cm}^2$ ,

**Plates:**  $\sigma_{t all} = 1.4 \text{ ton/cm}^2$  and  $\sigma_{bearing all} = 1.6 \text{ ton/cm}^2$



### Solution:

1. Failure due to bolt shear:

$$\tau_{all} \geq \frac{Q}{n(\pi d^2/4)} \rightarrow 1.1 \geq \frac{9}{(6 \times 2)(\pi d^2/4)} \rightarrow d \geq 0.932 \text{ cm} \quad \text{----- (1)}$$

2. Bearing failure of bolt or plate (compression failure):

$$\sigma_{bearing all} \geq \frac{Q}{n(dt)} \rightarrow 1.6 \geq \frac{9}{6(d \times 1.2)} \rightarrow d \geq 0.781 \text{ cm} \quad \text{----- (2)}$$

3. Plate tearing:

$$\begin{aligned} \sigma_{t all} \geq \frac{Q}{(w - nd)t} &\rightarrow 1.4 \geq \frac{9}{(10 - 2d) \times 1.2} \rightarrow 10 - 2d \geq \frac{9}{1.4 \times 1.2} \\ &\rightarrow -2d \geq \frac{9}{1.4 \times 1.2} - 10 \rightarrow -2d \geq -4.64 \rightarrow d \leq 2.32 \text{ cm} \quad \text{----- (3)} \end{aligned}$$

**From (1)-(3), the safe diameter of bolts is  $\geq 0.932 \text{ cm}$  and  $\leq 2.32 \text{ cm}$**

**the safe diameter of bolts is  $0.932 \text{ cm} = 9.32 \text{ mm}$  (take  $d = 10 \text{ mm}$ )**

### Check

$$\tau_{all} = \frac{Q}{n(\pi d^2/4)} = \frac{9}{12(\pi 1^2/4)} = 0.955 < 1.1 \text{ ton/cm}^2 \text{ O.K.}$$

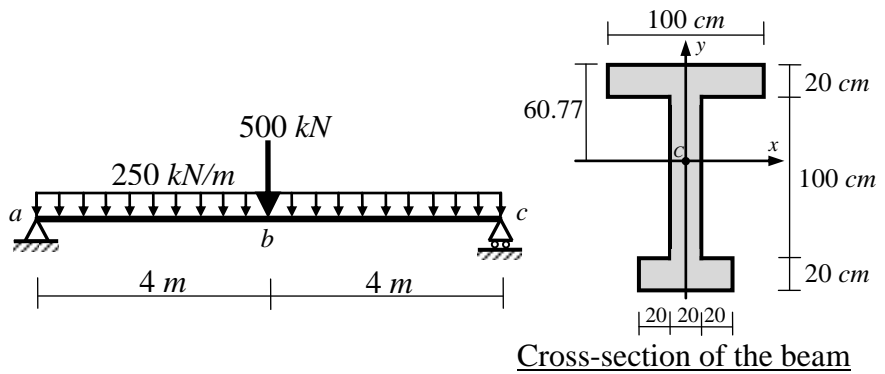
$$\sigma_{bearing all} = \frac{Q}{n(dt)} = \frac{9}{6(1 \times 1.2)} = 1.25 < 1.6 \text{ ton/cm}^2 \text{ O.K.}$$

$$\sigma_{t all} = \frac{Q}{(w - nd)t} = \frac{9}{(10 - 2 \times 1) \times 1.2} = 0.94 < 1.4 \text{ ton/cm}^2 \text{ O.K.}$$

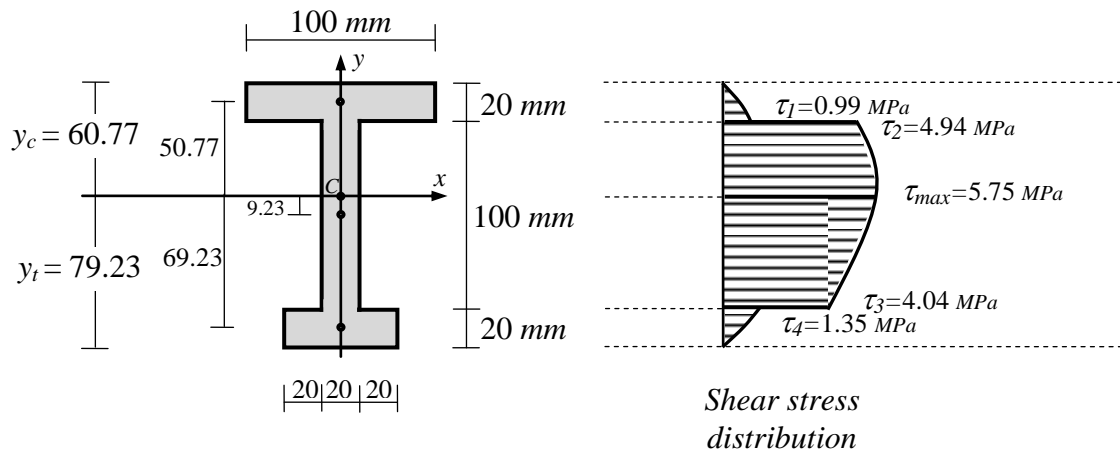
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**Question (3): (15 Marks)**

(a) For the shown beam, calculate and draw the shear stress distribution over the cross-section at *a*.



**Solution:**



$$I_x = [60 \times 20^3 / 12 + (1200)(-69.23)^2] + [20 \times 100^3 / 12 + (2000)(-9.23)^2] + [100 \times 20^3 / 12 + (2000)(50.77)^2]$$

$$= 12850256 \text{ cm}^4 = 12.85 \times 10^6 \text{ cm}^4$$

Shear force at *a*,  $Q_a = wL/2 + P/2 = 250 \times 8/2 + 500/2 = 1000 + 250 = 1250 \text{ kN} = 1.25 \times 10^6 \text{ N}$

Shear stress,  $\tau_1 = (Q_a S_x) / (I_x b) = (1.25 \times 10^6 \times 2000 \times 50.77) / (12.85 \times 10^6 \times 100)$   
 $= 98.77 \text{ N/cm}^2 = 0.99 \text{ MPa}$

$\tau_1 = 0.99 \text{ MPa}$

Shear stress,  $\tau_2 = (1.25 \times 10^6 \times 2000 \times 50.77) / (12.85 \times 10^6 \times 20)$   
 $= 493.87 \text{ N/cm}^2 = 4.94 \text{ MPa}$

$\tau_2 = 4.94 \text{ MPa}$

Shear stress,  $\tau_{max} = (1.25 \times 10^6)(2000 \times 50.77 + 20 \times 40.77^2 / 2) / (12.85 \times 10^6 \times 20)$   
 $= 574.72 \text{ N/cm}^2 = 5.75 \text{ MPa}$

$\tau_{max} = 5.75 \text{ MPa}$

Shear stress,  $\tau_3 = (1.25 \times 10^6)(1200 \times 69.23) / (12.85 \times 10^6 \times 20)$   
 $= 404.1 \text{ N/cm}^2 = 4.04 \text{ MPa}$

$\tau_3 = 4.04 \text{ MPa}$

Shear stress,  $\tau_4 = (1.25 \times 10^6)(1200 \times 69.23) / (12.85 \times 10^6 \times 60)$   
 $= 134.689 \text{ N/cm}^2 = 1.35 \text{ MPa}$

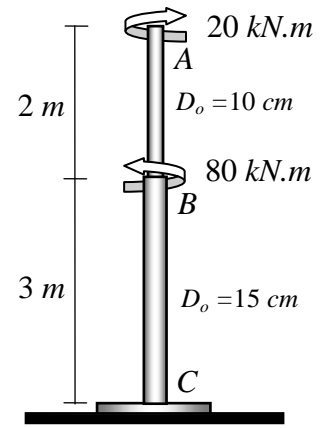
$\tau_4 = 1.35 \text{ MPa}$

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**Question (3): (15 Marks)**

(b) A column of two tubes must resist torques as shown. The tubes have outside diameters of 10 cm and 15 cm and a thickness of 10 mm.

- Draw the twisting moment diagram.
- Determine the maximum shear stress  $\tau_{\max}$  in the two tubes.
- Determine the relative angle of twist  $\phi$  between A and C, where  $G = 30 \text{ GPa}$



**Solution:**

The twisting moment diagram is as shown

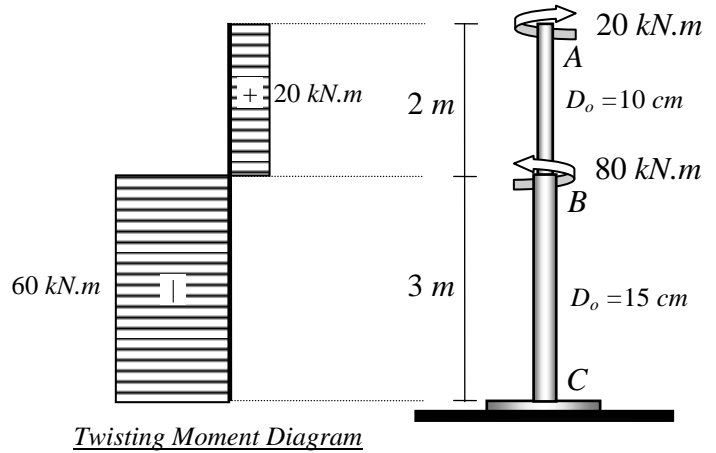
- The twisting moment diagram is as shown.

**Part AB**

Twisting moment,  $T = 20 \text{ kN.m} = 20 \times 10^6 \text{ N.mm}$

$$J = \frac{\pi(r_o^4 - r_i^4)}{2} = \frac{\pi(50^4 - 40^4)}{2} = 5.796 \times 10^6 \text{ mm}^4$$

$$\text{So, } \tau_{\max} = \frac{Tr_o}{J} = \frac{(20 \times 10^6)(50)}{5.796 \times 10^6} = 172.5 \text{ N/mm}^2 = \boxed{172.5 \text{ MPa}}$$



**Part BC**

Twisting moment,  $T = 60 \text{ kN.m} = 60 \times 10^6 \text{ N.mm}$

$$J = \frac{\pi(r_o^4 - r_i^4)}{2} = \frac{\pi(75^4 - 65^4)}{2} = 21.661 \times 10^6 \text{ mm}^4$$

$$\text{So, } \tau_{\max} = \frac{Tr_o}{J} = \frac{(60 \times 10^6)(75)}{21.661 \times 10^6} = 207.7 \text{ N/mm}^2 = \boxed{207.7 \text{ MPa}}$$

$$\phi_{A/C} = \phi_{A/B} + \phi_{B/C} = \frac{20 \times 10^6 \times 2000}{5.796 \times 10^6 \times 30000} + \frac{-60 \times 10^6 \times 3000}{21.661 \times 10^6 \times 30000} = 0.23 - 0.277 = 0.047 \text{ rad} = \boxed{2.7^\circ}$$

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