

## Answer of Final Exam

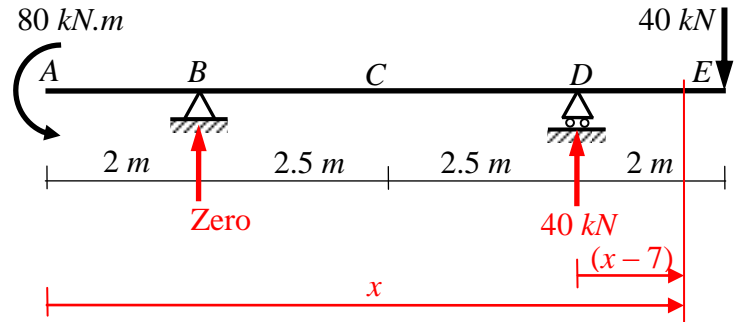
Total Marks: 70

No. of Questions: 5

### Question (1): (14 Marks)

For the shown beam, using the **double integration method**, determine:

- (a) the deflections at A, C and E  
(b) the slopes at A and C  
and sketch the elastic curve of the beam.  
 $EI = 5 \times 10^4 \text{ kN.m}^2$



#### Reactions:

$$\begin{aligned} \sum M_D &= -80 + B_y(5) + 40 \times 2 = 0 \rightarrow B_y = 0 \\ \uparrow F_y &= D_y - 40 = 0 \rightarrow D_y = 40 \text{ kN} \uparrow \end{aligned}$$

$$\begin{aligned} M &= -80(x)^0 + 40(x-7) \\ EIy'' &= -80(x)^0 + 40(x-7) \\ EIy' &= -80x + 20(x-7)^2 + C_1 \\ EIy &= -40x^2 + 20(x-7)^3/3 + C_1x + C_2 \end{aligned}$$

#### Boundary Conditions:

$$\begin{aligned} \text{At } x = 2 \text{ m, } y = 0 &\rightarrow 0 = -40(2)^2 + C_1(2) + C_2 \rightarrow 2C_1 + C_2 = 160 \\ \text{At } x = 7 \text{ m, } y = 0 &\rightarrow 0 = -40(7)^2 + 20(0)^3/3 + C_1(7) + C_2 \rightarrow 7C_1 + C_2 = 1960 \\ &\rightarrow \underline{C_1 = 360} \text{ and } \underline{C_2 = -560} \end{aligned}$$

So, the general equation of the deflection  $y$  at any distance  $x$  is,

$$EIy = -40x^2 + 20(x-7)^3/3 + 360x - 560$$

#### (a) The deflection at A ( $x = 0$ )

$$\begin{aligned} EIy_A &= -40(0)^2 + 20(0-7)^3/3 + 360(0) - 560 = -560 \\ y_A &= -560/50000 = -0.0112 \text{ m} = -11.2 \text{ mm} \end{aligned}$$

$$y_A = 11.2 \text{ mm} \downarrow$$

#### The deflection at C ( $x = 4.5$ m)

$$\begin{aligned} EIy_C &= -40(4.5)^2 + 20(4.5-7)^3/3 + 360(4.5) - 560 = 250 \\ y_C &= 250/50000 = 0.005 \text{ m} = 5 \text{ mm} \end{aligned}$$

$$y_C = 5 \text{ mm} \uparrow$$

#### The deflection at E ( $x = 9$ m)

$$\begin{aligned} EIy_E &= -40(9)^2 + 20(9-7)^3/3 + 360(9) - 560 = -1520/3 = -506.6667 \\ y_E &= -1520/3/50000 = -0.010133 \text{ m} = -10.13 \text{ mm} \end{aligned}$$

$$y_E = 10.1 \text{ mm} \downarrow$$

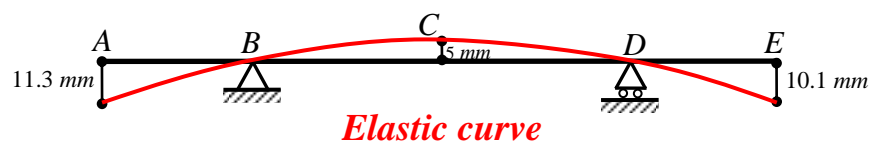
#### (b) The slope at A ( $x = 0$ )

$$\begin{aligned} EIy'_A &= -80x + 20(x-7)^2 + C_1 = 0 + 0 + 360 = 360 \\ y'_A &= 360/50000 = \underline{0.0072 \text{ rad}} \quad \downarrow = 0.41^\circ \end{aligned}$$

#### The slope at C ( $x = 4.5$ m)

$$\begin{aligned} EIy'_C &= -80x + 20(x-7)^2 + C_1 = -360 + 0 + 360 = 0 \\ y'_A &= 0/50000 = \underline{0} \end{aligned}$$

The slope at E ( $x = 9$  m)  
 $EIy'_E = -80x + 20(x-7)^2 + C_1 = -720 + 80 + 360 = 0$   
 $y'_E = -280/50000 = \underline{0.0056 \text{ rad}} \quad \downarrow$



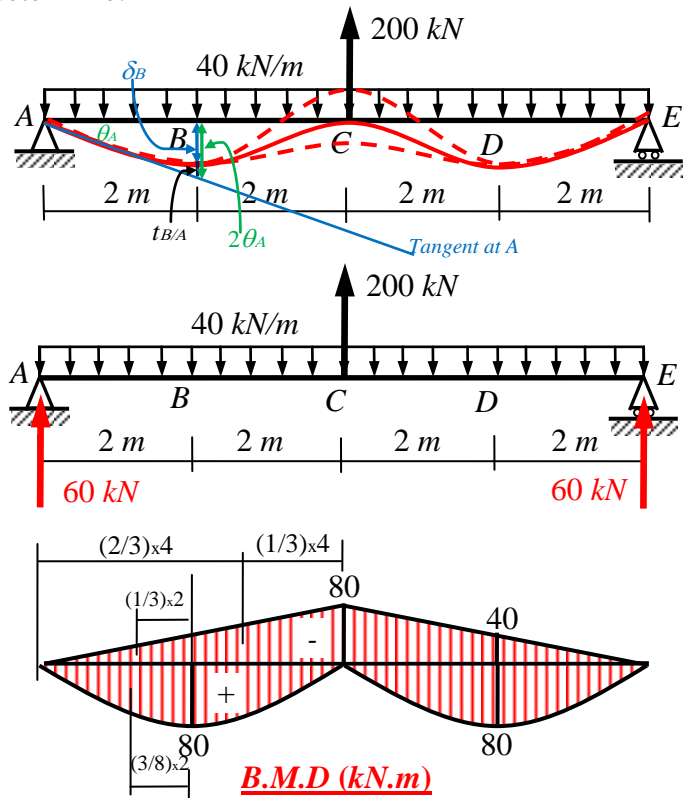
With my best wishes

Dr. M. Abdel-Kader

**Question (2): (14 Marks)**

For the shown beam, using the **moment-area method**, determine:

- (a) the slope at **A**  
 (b) the deflections at **C** and **B**  
 and sketch the elastic curve of the beam.  
 $EI = 1 \times 10^4 \text{ kN.m}^2$



- (a) From symmetry, the slope at **C** ( $\theta_C$ ) is equal to zero, the change in slope between the tangents of the elastic curve at points **A** and **C** ( $\theta_{AC}$ ) is equal to the slope at **A** ( $\theta_A$ ),

$$\theta_{AC} = \theta_C - \theta_A = 0 - \theta_A = -\theta_A$$

then

$$-\theta_A = \frac{1}{EI} [\text{Area}_{AC}]$$

$$= \frac{1}{10000} \left[ \frac{2}{3} \times 4 \times 80 - \frac{1}{2} \times 4 \times 80 \right] = \frac{160}{3 \times 10000}$$

$$= 0.00533 \text{ rad}$$

$$\therefore \theta_A = -0.00533 \text{ rad}$$

- (b) **The deflection at C** ( $\delta_C$ ) =  $4\theta_A - t_{C/A}$

$$t_{C/A} = \frac{1}{EI} [\text{Area}_{AC} \cdot \bar{X}_C] = \frac{1}{10000} \left[ \left( \frac{2}{3} \times 4 \times 80 \right) (2) - \left( \frac{1}{2} \times 4 \times 80 \right) \left( \frac{1}{3} \times 4 \right) \right] = 8/375 = 0.02133 \text{ m}$$

$$\delta_C = 4\theta_A - t_{C/A} = 4(0.00533) - 0.02133 =$$

$$\therefore \delta_C = 0$$

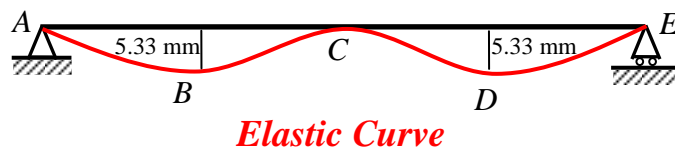
OR

$$\delta_C = t_{A/C} = \frac{1}{EI} [\text{Area}_{AC} \cdot \bar{X}_A] = \frac{1}{10000} \left[ \left( \frac{2}{3} \times 4 \times 80 \right) (2) - \left( \frac{1}{2} \times 4 \times 80 \right) \left( \frac{2}{3} \times 4 \right) \right] = 0 \quad \therefore \delta_C = 0$$

- The deflection at B** ( $\delta_B$ ) =  $2\theta_A - t_{B/A}$

$$t_{B/A} = \frac{1}{EI} [\text{Area}_{AB} \cdot \bar{X}_B] = \frac{1}{10000} \left[ \left( \frac{2}{3} \times 2 \times 80 \right) \left( \frac{3}{8} \times 2 \right) - \left( \frac{1}{2} \times 2 \times 40 \right) \left( \frac{1}{3} \times 2 \right) \right] = 2/375 = 0.00533$$

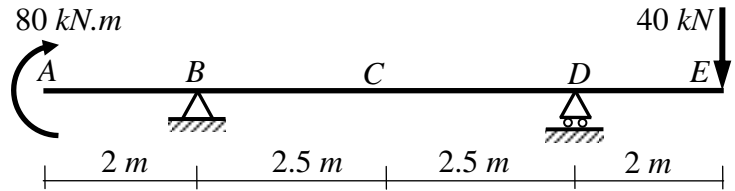
$$\delta_B = 2\theta_A - t_{B/A} = 2(0.00533) - 0.00533 = 0.00533 \text{ m} = 5.33 \text{ mm} \downarrow \quad \therefore \delta_B = 5.33 \text{ mm} \downarrow$$



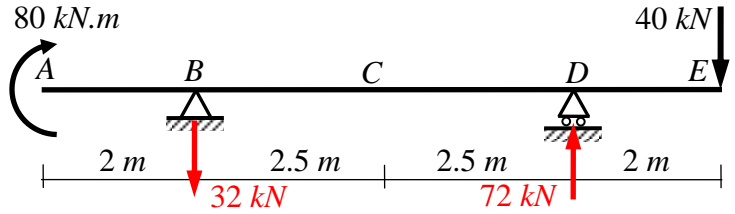
**Question (3): (14 Marks)**

For the shown beam, using the **conjugate beam method**, determine:

- (a) the deflections at **A**, **C** and **E**
  - (b) the slopes at **A** and **C**
- and sketch the elastic curve of the beam.  
 $EI = 5 \times 10^4 \text{ kN.m}^2$



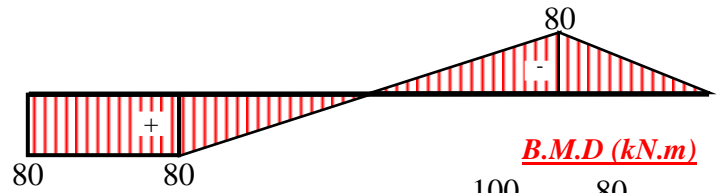
The bending moment diagram is as shown. The resulting moment diagram is then loaded to the conjugate beam. For the conjugate beam, determine the elastic reaction ( $r_A$  and  $r_B$ ).



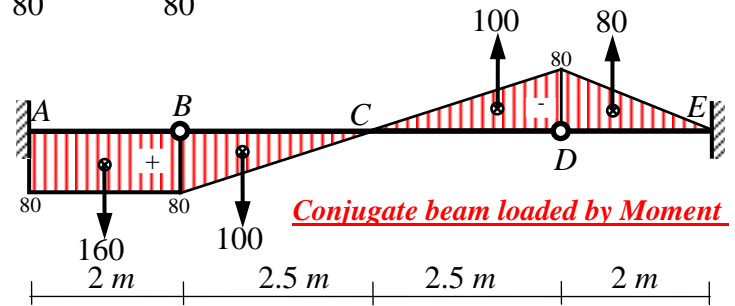
To determine the elastic reaction at **B** ( $R_B$ ) take the moment about **D** for part **BD**:

$$\rightarrow R_B = 200/3 \uparrow$$

Then  $R_D = 200/3 \downarrow$

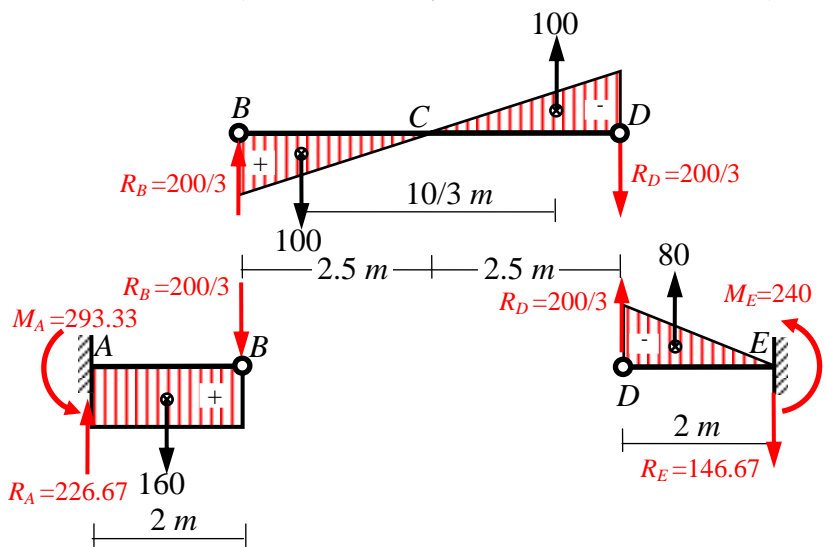


**Deflection at A**  $= M_A / EI = -293.33 / 50000$   
 $\delta_A = -293.33 / 50000 = -0.005867 \text{ m}$   
 $\therefore \delta_A = 5.87 \text{ mm} \uparrow$



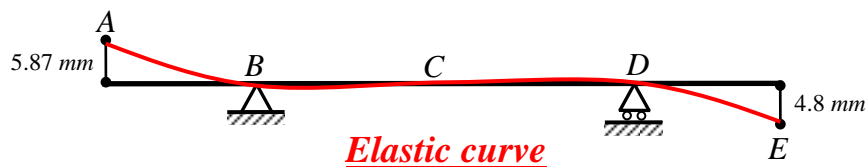
**Deflection at C**  $= M_C / EI$   
 $= [200/3(2.5) - 100(5/3)] / 50000$   
 $\delta_C = 0 \quad \therefore \delta_C = 0$

**Deflection at E**  $= M_E / EI = 240 / 50000$   
 $\delta_E = 240 / 50000 = -0.0048 \text{ m}$   
 $\therefore \delta_E = 4.8 \text{ mm} \uparrow$



**Slope at A**  $= \text{Shear at A} / EI = 226.67 / 50000$   
 $= 0.00453 \quad \theta_A = 0.00453 \text{ rad}$

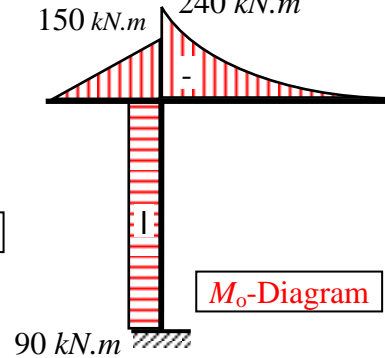
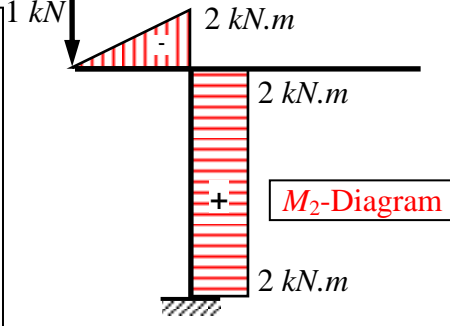
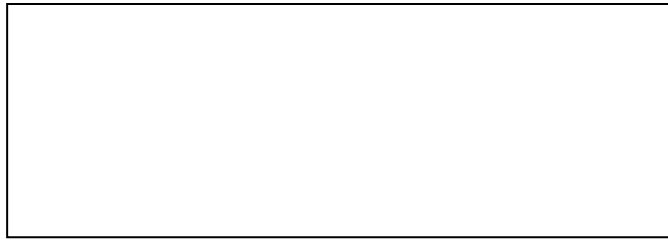
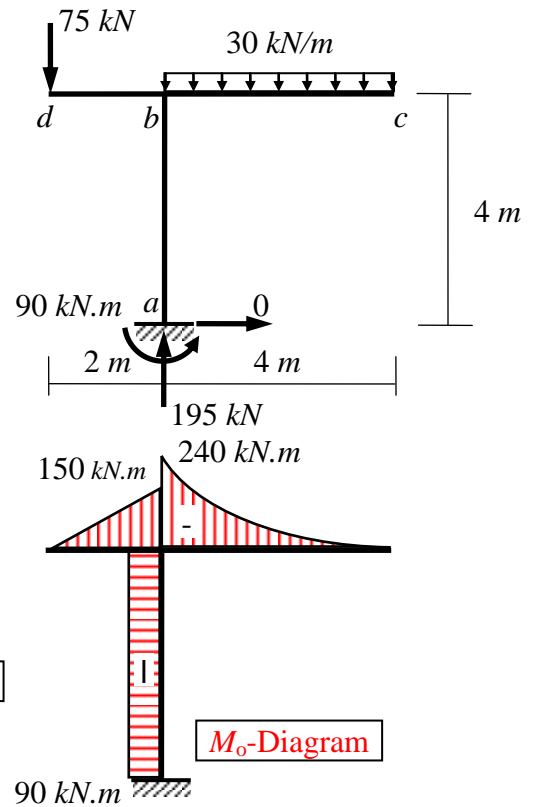
**Slope at C**  $= \text{Shear at C} / EI = [200/3 - 100] / 50000$   
 $= [-33.33] / 50000 = -0.00067 \quad \theta_C = -0.00067 \text{ rad}$



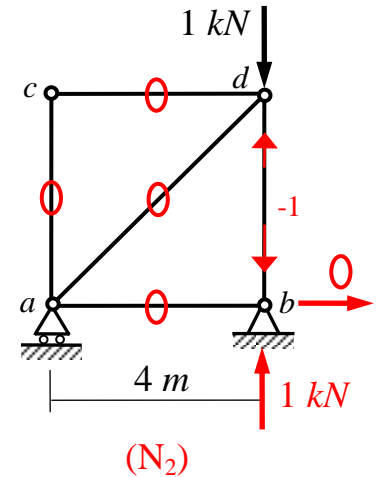
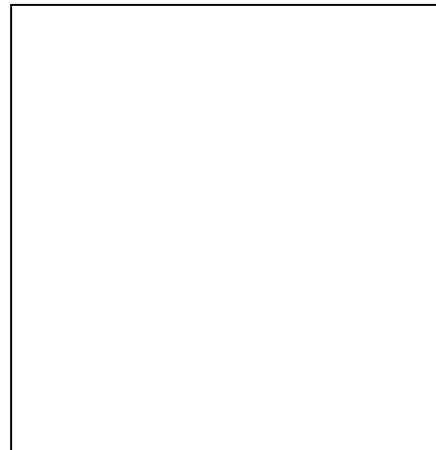
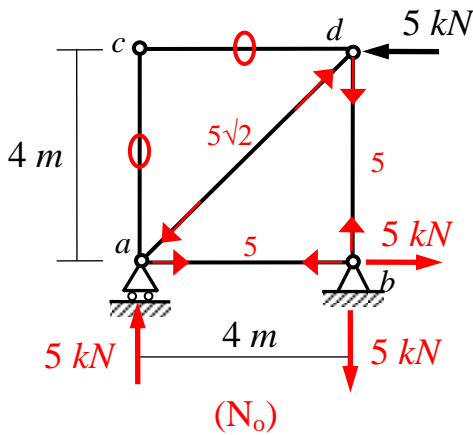
**Question (4): (12 Marks)**

For the shown frame and truss, using the **virtual work method**, determine the vertical displacement at  $d$  ( $\delta_{dv}$ ).

For the frame, the relative moments of inertia are given between brackets and  $EI=20 \times 10^3 \text{ kN.m}^2$ . For the truss, assume that all members have the same axial rigidity  $EA=1000 \text{ kN}$ .



$$\delta_{dv} = \int \frac{M_o M_2}{EI} dL = \frac{1}{EI} [(-4 \times 90)(2)] + \frac{1}{EI} [(-\frac{1}{2} \times 2 \times 150)(-\frac{2}{3} \times 2)] = \frac{-520}{EI} \therefore \delta_{dv} = -\frac{520}{20000} = 0.026 \text{ m} = 26 \text{ mm} \uparrow$$



Reactions and member forces due to:

(a) Applied load

(b) Horizontal unit load at  $d$

(c) Vertical unit load at  $d$

Calculation details for horizontal and vertical deflection of joint  $d$

Member	EA (kN)	L (m)	$N_o$ (kN)	$N_2$ (kN)	$N_o N_2 L / EA$
1(ac)	1000	4	0	0	0
2(cd)	1000	4	0	0	0
3(bd)	1000	4	5	-1	-0.02
4(ab)	1000	4	5	0	0
5(ad)	1000	$4\sqrt{2}$	$-5\sqrt{2}$	0	0
$\Sigma$					$\delta_{dv} = -0.02 \text{ m}$ (= 20 mm Upwards)



$$\text{Vertical deflection of joint } d = \delta_d^v = \sum \frac{N_o N_2 L}{EA} = 20 \text{ mm} \uparrow$$

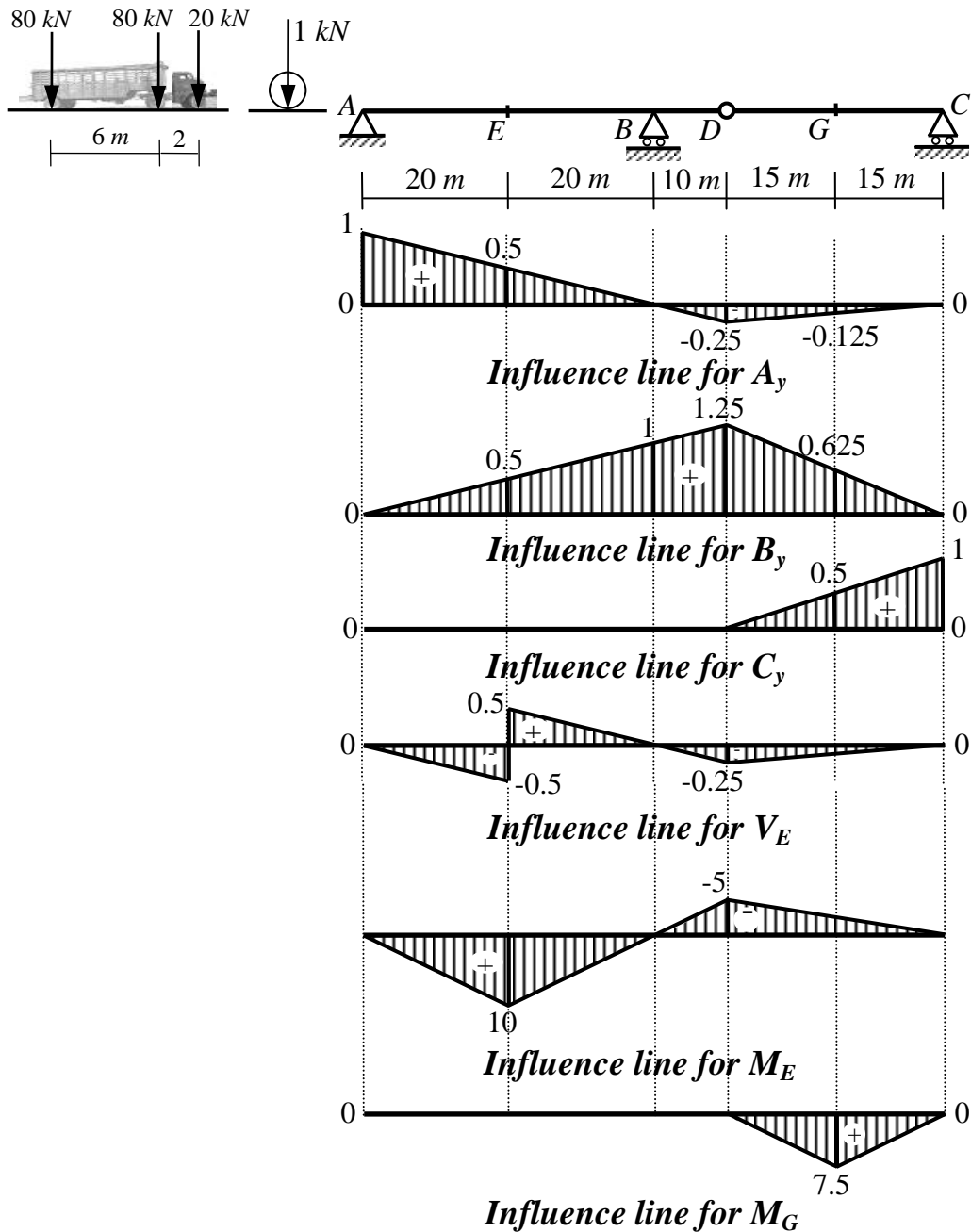
**Question (5): (12 Marks)**

For the shown beam, draw the influence line for:

(a) the reactions  $A_y$ ,  $B_y$  and  $C_y$ .

(b) the shear force at the section  $E$  and the bending moments at the sections  $E$  and  $G$ .

Also, determine the maximum moment at  $G$  caused by the shown moving truck.



$$M_{G \max} = 80(4.5) + 80(7.5) + 20(6.5)$$

$$= 1090 \text{ kN.m } \uparrow \uparrow$$

$$\therefore M_{G \max} = 1090 \text{ kN.m } \uparrow \uparrow$$

